

NO-A188 025 CRYOCOOLER SYSTEMS DEMONSTRATION(U) TRW SPACE AND  
TECHNOLOGY GROUP REDONDO BEACH CA R D SANDELL ET AL  
31 AUG 87 N00014-84-C-0397

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FINAL REPORT

on

CRYOCOOLER SYSTEMS DEMONSTRATION

Contract No. N00014-84-C-0397  
TRW Project No. 44236

Period-of-Performance  
4 September 1984 - 23 February, 1986

Submitted to

The Office of Naval Research  
800 N. Quincy St.  
Arlington, VA 22217

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31 August, 1987

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## SUMMARY

This is the Final Report on Contract No. N00014-84-C-0397 on the integration and evaluation of a small plastic Stirling-cycle cryocooler and SQUID gradiometer. TRW successfully transferred Stirling-cycle cryocooler technology from its development at NBS to industry. We installed, modified, and operated a small, plastic, five-stage Stirling cycle refrigerator which was developed at the National Bureau of Standards and furnished to TRW for this project. We also designed and fabricated a new gas handling system, pressure wave generator and displacer drive, and fiber-glass-epoxy sleeves for another cryocooler. We successfully designed, fabricated, and operated a monolithic thin film SQUID gradiometer on a single silicon chip. The cooler was modified by the addition of a 4 kelvin Joule-Thompson stage before delivery to TRW and we initially fabricated this SQUID with a Nb-Pb technology for 4 kelvin experiments. We later transferred to an all-niobium technology developed at TRW when it was decided to operate the cryocooler at about 7 kelvin without the Joule-Thomson stage. We designed, constructed, and installed all components necessary to interface the magnetometer with the cryocooler. The silicon chip magnetometer was mounted on a silicon printed circuit board which was heat-sunk to the final stage of the cryocooler. Radiation shields were constructed and low noise and low thermal electrical leads were provided.

## 1. Introduction

The objectives of this program were to design, fabricate, and integrate a thin film SQUID gradiometer with a small plastic Stirling cryocooler and evaluate the performance. This required the transfer of small closed-cycle cryogenic refrigerator technology from the laboratory to industry and evaluate an operational cryocooler/magnetometer system. We designed and fabricated a monolithic thin film dc SQUID magnetometer, interfaced the magnetometer to the Stirling closed-cycle cryocooler built and furnished by the National Bureau of Standards (NBS). The inductance of the two-junction, dc SQUID is split into two parallel loops and is insensitive to a uniform field applied perpendicular to the plane loops. However it responds to a difference in field applied to the two coils (dHz/dx). In order to increase the magnetic field gradient sensitivity, superconducting dc transformers were used to couple flux from large (9mm by 3mm) input pick-up loops to the small SQUID inductance. The magnetometer was designed to fit on a one centimeter square silicon chip.

We successfully installed and operated the five stage Stirling-cycle helium refrigerator developed at NBS. The cryocooler uses two pneumatically driven diaphragm compressors which operate on the He gas. These diaphragm compressors, the pressure wave generator (PWG) and the displacer drive (DD), are controlled by solenoid valves which determine the timing and duration of their strokes. A Commodore 64 computer was chosen as an inexpensive, versatile controller to operate the cryocooler valves. Through a suitable interface the binary output of the computer operates the solenoid valves and thus the frequency of operation and the timing of the phases in the cycle. Timing could easily be changed in software by changing a byte in the memory.

The cryocooler was operated continuously for longer than three weeks on several occasions and achieved a minimum temperature below 7 kelvin. However, the operating temperature was neither reproducible nor extremely stable. The cryocooler required frequent fine adjustment of the manual needle valves to obtain optimum operation.

The cryocooler received from NBS had an additional helium gas Joule-Thompson stage in order to reach 4 K and be able to operate Pb-alloy based Josephson devices. Because of the difficulty in operating the miniature J-T valve successfully, we removed the J-T from the cryocooler and switched from the Nb-Pb alloy to an all-Nb process for fabricating the SQUID which could then operate at higher temperatures.

As a part of this program, we also designed and fabricated new components for the pressure wave generator and the displacer drive, and the fiberglass-epoxy sleeves for another cryocooler assembly. These components were designed in consultation with Dr. J. E. Zimmerman at NBS, who designed and built the first

cryocooler. We also designed and constructed an entirely new compressor/gas handling system.

## 2. SQUID Magnetometer

### 2.1 SQUID Design

Figure 1 is a schematic diagram of the planar thin film gradient dc SQUID which we designed and constructed. Figure 2 is the chip layout and Fig. 3 is a photograph of the monolithic gradient SQUID. The two junction SQUID has its inductance split into two parallel loops ( $L_S$ ). It is insensitive to a uniform field perpendicular to the loops since currents induced in the two SQUID inductances cancel each other at the junctions. However, it responds to a difference in field in the loops ( $dH/dx$ ). The SQUID inductances are very small, so to increase the field sensitivity, superconducting dc transformers are used to couple flux from the input pick-up loops ( $L_P$ ) to the SQUID. The magnetometer fits on a one centimeter square silicon chip.

Each pick-up loop is 9mm by 3mm, which gives an inductance of 24 nH. To maximize the flux coupled to the SQUID, the pickup loop inductance and the primary inductance of the flux transformer, ( $L_i$ ) should be equal. In that case, half of the coupled flux is transferred to the SQUID. To accomplish this and strongly couple the input inductance to the SQUID inductance, a 21 turn coil flux transformer was used. This inductor is made of 5  $\mu$ m wide microstrip tightly coupled to the SQUID with a calculated L of 24 nH. The SQUID inductor is formed by a Nb thin film with a 5  $\mu$ m wide slot with a square 30 $\mu$ m x 30 $\mu$ m hole at each end. Each hole has an inductance of 50 pH, giving a total SQUID inductance of 25 pH plus the parasitic inductance of the slot. To reduce the parasitics, a superconducting flap, insulated on one side, covers the slot. However the input coil tightly coupled to the SQUID reduces the SQUID inductance further. Assuming a coupling constant  $k = 0.8$ , we calculate the SQUID inductance at 15 pH. The critical current was designed for about 40  $\mu$ A with a beta less than 2, and the junctions were damped with 1 ohm resistors to eliminate hysteresis in the I-V characteristics. All current and voltage leads are brought in symmetrically on the top and bottom of the SQUID. The gate current is fed in on coplanar type lines and external flux bias leads to the pick-up loops are microstrip lines to decrease magnetic coupling of bias currents to the SQUID.

### 2.2 SQUID Fabrication

The first thin-film SQUIDS were made using Nb base-electrode junctions with a lead alloy counter-electrode. The following six layer process was used:

1. First, a Nb film was patterned to form the SQUID inductor, base electrode, pick-up loops, current bias and voltage leads.
2. SiO was deposited to define the junctions and insulate part of the SQUID.
3. Resistors were deposited to shunt the junctions.
4. The exposed Nb was oxidized to form the junctions, and Pb alloy counter-electrodes were deposited. This film also formed flaps over the slot in the SQUID inductance to reduce the



parasitic inductance which is not coupled to the input coil.

5. SiO was deposited to insulate the input coil from the SQUID.

6. Nb was deposited for the input coil and wiring interconnections.

After a number of unsuccessful attempts and considerable effort to operate the Joule-Thompson valve, we decided to make an all-niobium SQUID and mount it on the final stage of the Stirling refrigerator and operate around 7 K. The all Nb circuit technology was not initially available, but was developed during the contract with other TRW funds. To switch to an all Nb circuit, we changed two masks and used a tri-layer selective etching process to fabricate the Nb circuits. A Nb-Al<sub>2</sub>O<sub>3</sub>-Nb tri-layer junction covered the whole wafer and was patterned as in #1 above. Then junctions were defined by reactive plasma etching. Next a single SiO layer was deposited with vias to contact the junctions. The circuit was then completed with the resistor and the Nb wiring layers.

### 2.3 SQUID Performance

A representative I-V curves with different applied fields is shown in Fig. 4(a), and the corresponding voltage modulation with fixed gate current in Fig. 4(b). The field current was applied to the center of one of the input transformers. Assuming that the current divides equally between the pick-up loop  $L_p$  and the input inductor  $L_i$ , the mutual inductance of the input coil and the SQUID is 0.5 nH. If the current is applied directly to the SQUID, the SQUID film inductance obtained from the flux period is 10 pH. This implies a coupling constant  $k$  equal to 0.7. The all Nb SQUIDS which didn't have the flap to reduce the parasitic inductance had an inductance nearly twice as large.

### 2.4 Magnetometer Cryocooler Interface

When the magnetometer was cooled in the Stirling cycle refrigerator, the SQUID silicon chip was mounted on a printed circuit board as shown in Fig. 5. This board was also made of silicon for good heat transfer to the final stage of the refrigerator. The board was mechanically clamped with heat-sink compound to a copper block threaded into the final stage of the Stirling cycle refrigerator as shown in Fig. 6. Electrical leads were attached to the chip by wire bonding and to the board by soldering. Low thermal conductivity manganin wire connected the board to room temperature. With an all Nb magnetometer installed in the cryocooler, we were able to reach the Nb transition temperature as evidenced by a zero voltage critical current of the SQUID. However the temperature was not low enough to operate the device as a magnetometer, because of the reduced current densities of the junctions.

### 3. Cryocooler

TRW successfully installed and operated the five stage Stirling cycle helium refrigerator developed at The National Bureau of Standards (NBS). The cryocooler uses two pneumatically driven diaphragm compressors which operate on the He working gas. These compressors, the pressure wave generator (PWG) and the displacer drive (DD) are controlled by solenoid valves which determine the timing and duration of their strokes. In addition, there are several manual metering valves in both the He and compressed air lines to control the amplitude of the pressure wave and the displacement.

We chose a Commodore 64 computer as an inexpensive yet very versatile controller to operate the cryocooler. The binary output of the computer operated the solenoid valves, and established the frequency of operation and the timing of the phases in the cycle. Any timing value could easily be changed by setting a byte in the memory. We also added an analog-to-digital board to monitor and record the temperature. This was used to try to optimize the refrigerator operation. Pressure transducers were installed in the PWG and DD lines as additional diagnostic support to the operation of the cryocooler. These waveforms could be recorded on an oscilloscope and compared with traces taken at other times.

We operated the cryocooler continuously for over three weeks on several occasions and achieved a minimum temperature below 7 kelvin. However, the operating temperature was neither reproducible nor extremely stable. The cryocooler required frequent re-adjustment of the manual needle valves to obtain optimum operation.

Figure 7 is the schematic diagram of the cryocooler. The PWG is controlled by the solenoid valves V1, V2 and V5. When V1 opens, compressed air expands the diaphragm and the helium gas in the refrigerator is compressed. The gas is expanded by opening the exhaust valves V3 and V5, relieving the pressure on the diaphragm. The movement of the displacer, which shuttles the gas between the warm and cold ends of the refrigerator at constant pressure, is actuated by the displacer drive and controlled by solenoid valves V2 and V4. Opening V2 drives the displacer piston up, and opening the exhaust valve V4 allows the displacer to move down.

The manual valves  $V_{m1}$  through  $V_{m5}$  are for fine tuning the system. Valve  $V_{m1}$  is a small leak to ensure equal average pressure in the refrigerator and the displacer drive. The gases on these two sides are isolated. The "working" gas of the refrigerator is in the pressure wave generator. Valves  $V_{m2}$  and  $V_{m3}$  to control the stroke of the displacer. Valve  $V_{m4}$  is used to maintain a constant average pressure in the cryocooler which can decrease due to leaks or cooling.

The seven phases of the valve sequences for one cycle of the Stirling cycle are described below. (Refer to Fig. 7 for the solenoid valve numbers). Figure 8 is a schematic diagram of the pressure in the refrigerator over a cycle.

1. When the displacer has nearly reached the bottom and the sinking gas is at the warm end of the cryocooler V1 opens, driving the diaphragm in the PWG and compressing the He gas in the refrigerator.

2. Exhaust valve V4 of the displacer drive closes, ending the downward movement of the displacer.

3. V1 closes, ending the compression, and V2 opens, moving the displacer up and transferring the compressed gas to the cold end.

4. V2 closes, ending the movement of the displacer and exhaust valve V3 opens, decreasing the pressure from the PWG, and the gas is expanded.

5. A second PWG exhaust valve V5, which is in parallel with V3 which opened in phase 4, also opens to ensure complete expansion.

6. Exhaust valve V4 of the DD opens, allowing the displacer to fall downward and transfer the expanded gas to the warm end.

7. Exhaust valves V3 and V5 of the PWG close in preparation for the next compression.

This cycle is then repeated.

### 3.1 Controller Operation

A Commodore 64 computer was used as the controller to open and close the compressed air solenoid valves which move the diaphragm on the displacer and pressure wave generator drives. All sequencing and timing was done by software. Five of the eight output bits from the User Port are used to drive 5V relays after being conditioned by a TTL hex inverting buffer. The relays operate the 110V ac solenoid valves. The main program to run the refrigerator is written in machine language and uses interrupts to gain control of the computer as shown in Fig. 9. By doing this, other programs may be run on the computer to optimize, read temperature, display valve status, etc., while the refrigerator timing control program is being run in the background. The program uses one of the internal timers to generate a non-maskable interrupt about every millisecond. When this happens the program takes control and checks if it is necessary to change the valves. A cycle consists of 5 phases of valves settings and corresponding times. The computer checks the current phase at 1 msec intervals to see if it is finished. If it isn't complete, control is returned. If it has timed out, the valve settings and time for the next phase are read from a table in memory and the new valve bits are sent to the output port

before returning control. The timing or valves for any phase can be changed by writing new bytes in the valve and timing tables (i.e. in BASIC POKE table,byte). The overall period of a cycle can be changed by changing how often an interrupt is generated. This is done by writing two bytes to the internal timer.

Programs written in BASIC can be run concurrently with the machine language program and perform tasks such as reading and displaying temperature, and optimizing the cooler by changing the valve timing.

Using a program written in BASIC, we have attempted to optimize the timing for these phases. The program reads the temperature of the coldest stage (carbon resistor), changes the timing of one phase at a time, and reads the change in temperature after about 30 to 60 minutes. When this has been done for all phases, the timing change which has made the best improvement becomes the new time for that phase and the process repeats. The difficulty in doing this is that the response time is very long and one iteration can take 10 to 20 hours. During this time the operation of the refrigerator often changes and fine tuning of the manual valves is needed. The best values found for the valve timing phases are:

- 1) 120 ms
- 2) 280 ms
- 3) 230 ms
- 4) 480 ms
- 5) 130 ms
- 6) 60 ms
- 7) 280 ms.

Using these parameters, a typical optimized temperature of the final stage was 7 K. The first four stages had nominal temperatures of 175, 75, 30 and 16 K. The refrigerator would operate continuously for about 2 to 3 weeks with some manual adjustments.

### 3.2 Cryocooler Modifications

Before operating the cooler, it was necessary to replace several defective valves in the original assembly with bellows sealed valves, and to machine two of the displacer stages and the displacer piston. The plastic displacer parts did not have sufficient clearance to operate smoothly when the cooler was assembled at TRW. This was evidently caused by swelling in the moist air when the system was disassembled. We subsequently found it helpful to out-gas the plastic parts under vacuum overnight before assembly whenever the system was disassembled for long periods of time.

As in all refrigerators, moisture is a major problem. Another problem is the diffusion of water molecules through the neoprene diaphragms in the PWG and DD. To minimize this source of moisture in the cooler, we installed an air drier on the compressed air input and also pumped on the space between the

double layered diaphragms.

NBS added a Joule-Thompson (J-T) stage to the final stage in order reach 4 K and operate Pb-alloy based Josephson devices. We were unable to operate the J-T valve satisfactorily. Because of the low flow rates required and thus the very small orifice, the valve clogged and could not be cleared. We fabricated several replacement throttling valves with no success in achieving the desired flow rates. A picture of one of the fabricated J-T stages is shown in Fig. 10. We decided to remove the J-T from the cryocooler and switch from the Nb-Pb to an all-Nb process for SQUID materials.

Under this program we also designed and fabricated new components for the pressure wave generator and the displacer drive. These components were designed in consultation with Dr. J. Zimmerman at NBS, who designed and built the cryocooler. We also designed and constructed an entirely new self-contained compressor/gas handling system. The gas handling valve and piping schematic is shown in Fig. 11. The system is housed in a chassis occupying about 4 cu ft as shown in the photograph in Fig. 12. Small cylinders for the compressed air ballast, high pressure He storage, low pressure He make-up reservoir, and He filter are housed in the bottom section of the unit. For easy access, the bellows sealed metering and shut off valves, regulators and pressure guages are mounted on the front panel. The only connections necessary for operation are dry compressed air, He for filling the storage cylinder, and vacuum for initially purging the system. These connections are located on the rear panel. Under Zimmerman's direction, we also fabricated the fiberglass-epoxy sleeves for another cryocooler assembly. Using a lathe, Kevlar cord was wound one layer at time on a mandrel coated with a mold release. After each layer, epoxy resin was brushed on and allowed to set slightly before continuing with the next layers.

### 3.3 Refrigerator Operational Procedures

1.) Purge: After He lines from the displacer drive and pressure wave generator are connected to the system, purge by pumping and flushing with ultra pure Helium gas numerous times (>10). Be sure all He line valves are fully open.

2.) Charge system with pure He gas to 40 psi.

3.) Turn on dry compressed air (80 psi) to displacer and PWG drives.

4.) Turn on controller (Commadore-64) to the solenoid air valves. These are 5 volt signals.

5.) Close the needle valve between the displacer drive and the pressure wave generator drive ( $V_{m1}$ ), and then open 1/2 turn. There should be practically no flow through this valve, it is only to assure equal average He pressure between the displacer and pressure wave generator drives.

6.) Close the valve between the two stages of the system until no pressure fluctuation is observed on the gauge. Again there should be no leakage. The system replenishes any He that might leak out of the system.

7.) Adjust the valve from the Joule-Thomson stage displacer ( $V_{m2}$ ) for smooth operation. When the system is warm, the motion is often erratic. As it cools, when the system is cold, the motion is turned down to prevent the displacer from hitting the bottom of the stroke. The final temperature is the setting of this valve.

8.) The cooling of the refrigerator is monitored by measuring first the upper stage thermopiles and then the lower ones as they begin to cool. It takes about 10 minutes to get cold, at which time the carbon resistors on the lower stage can be monitored. The stroke of the displacer is reduced by adjusting the manual valve on the line to the solenoid valve ( $V_{m3}$ ) on the displacer line.

#### 4. Conclusion

TRW successfully transferred the technology to industry to operate, and fabricate the small split-film refrigerator from NBS to industry. Performance was fully demonstrated, which was demonstrated at NBS. We also demonstrated that we successfully tested a planar thin film refrigerator using niobium technology. A difficulty in operation was the frequent requirement to manually adjust the system for optimum performance. For future use, we recommend that we monitor operating parameters such as the dynamic pressure of the displacer position under computer control and adjust these adjustments. The Joule-Thomson stage also needs to be feasible because of the difficulty of making and maintaining precision small orifice. We would like to acknowledge Zimmerman for his assistance in the design of new components for the operation of the cryocooler, and M. K. Wagner for the fabrication of these components.

### Figures

1. Schematic of thin-film gradient magnetometer.
2. Pen plot of the planar SQUID and input coils.
3. Photo of the magnetometer chip.
4. (a.) Flux modulation of the current-voltage curves of the SQUID.  
(b.) Voltage modulation vs. applied field.
5. Photo of silicon pc board with chip attached.
6. Sketch of the attachment of the pc board to the lower stage of the cryocooler.
7. Schematic of cryocooler.
8. Timing diagram of the cryocooler cycle.
9. Assembly language listing of the control program.
10. Photo of original (left) and new Joule-Thompson stages.
11. Schematic of new gas handling valve and piping.
12. Photo of compressor and valving assembly.

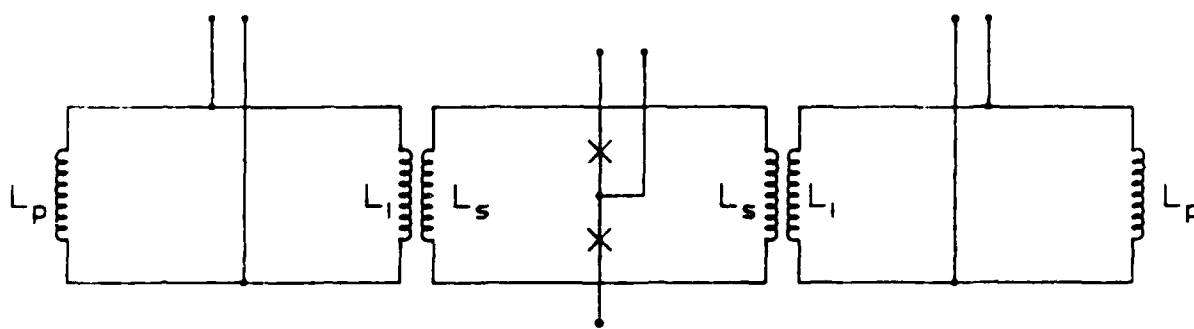


Figure 1. Schematic of thin-film gradient magnetometer.



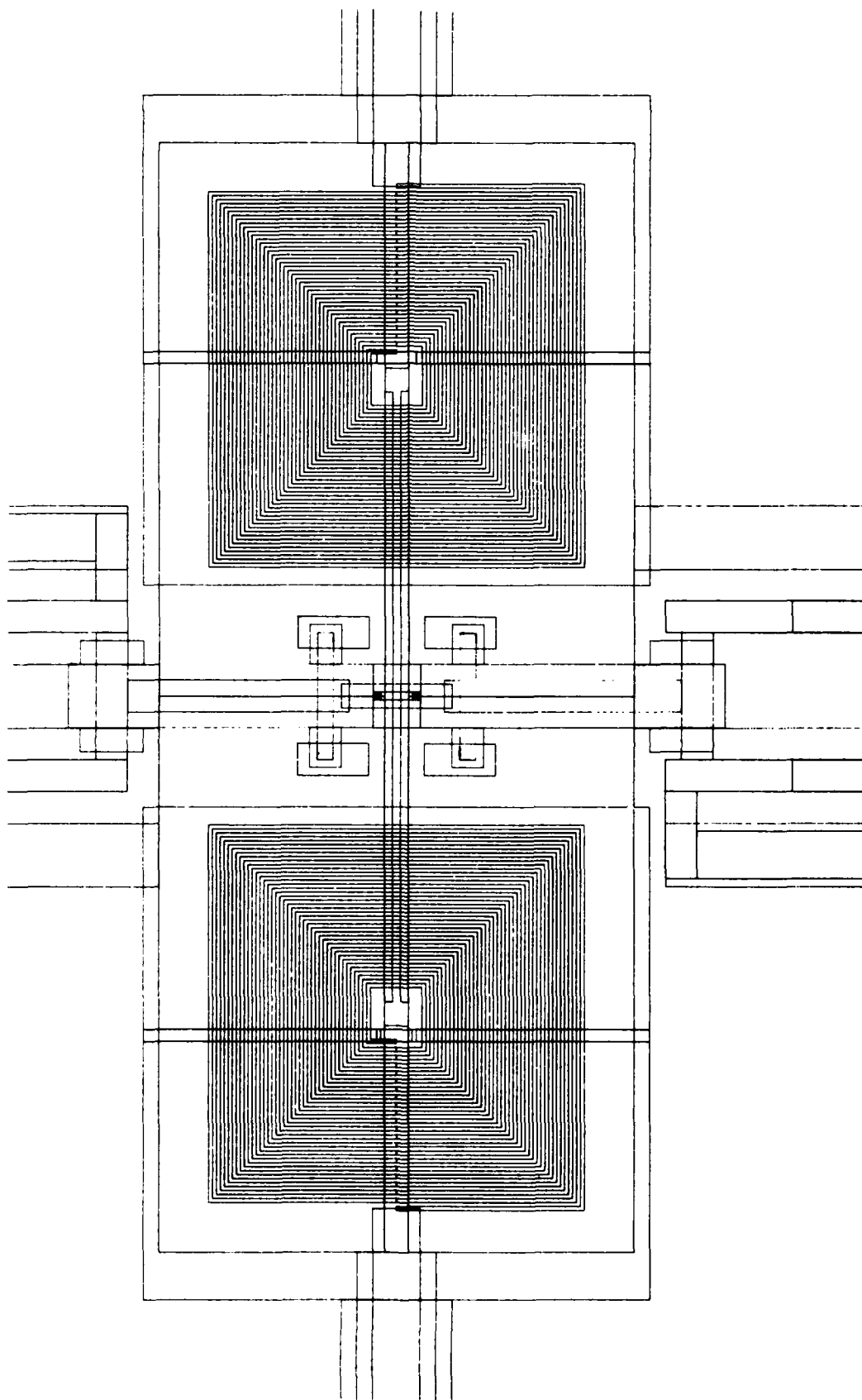


Figure 2. Pen plot of the planar SQUID and input coils.

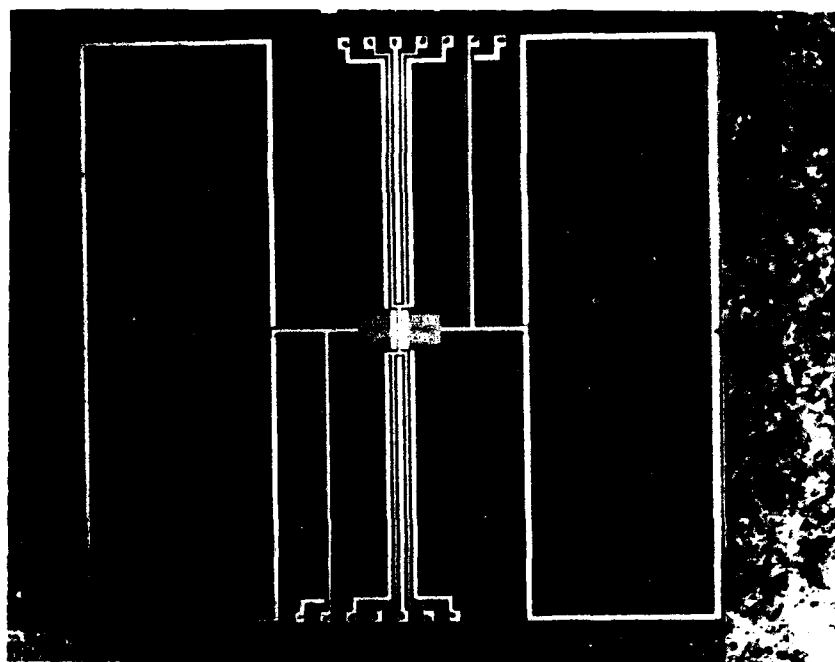


Figure 3. Photo of the magnetometer chip.

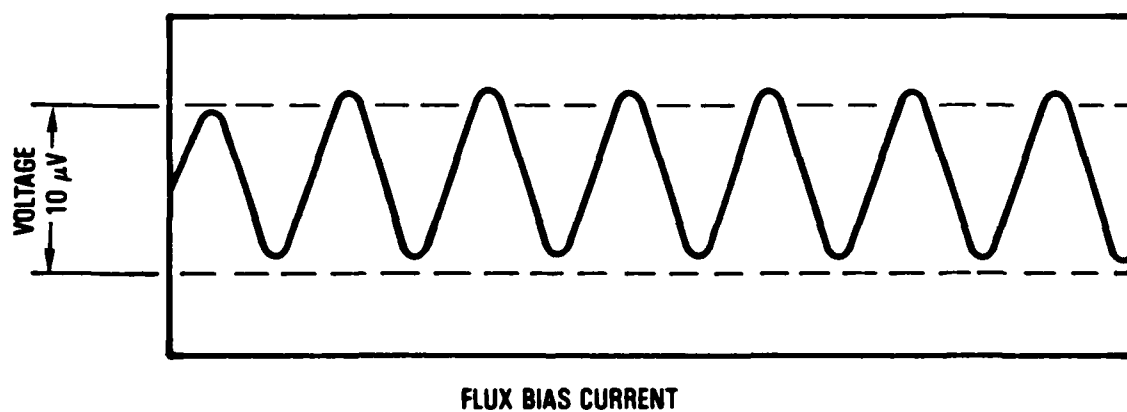
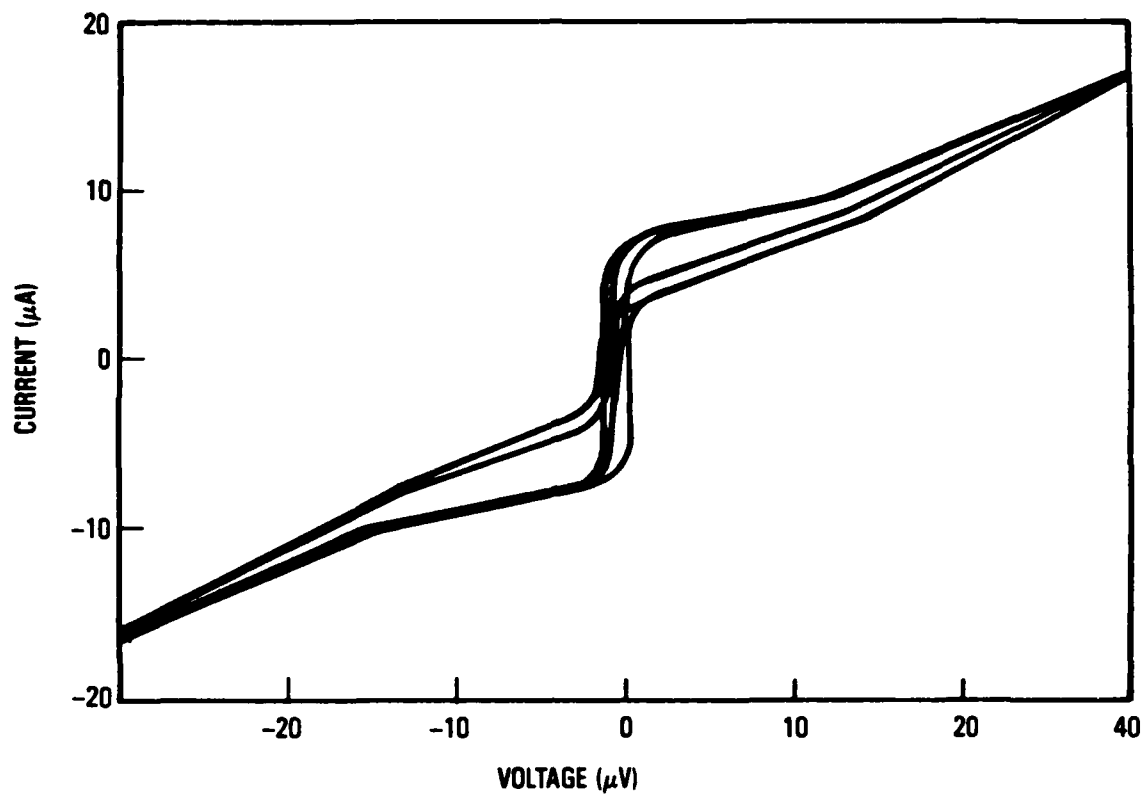


Figure 4. (a.) Flux modulation of the current-voltage curves of the SQUID. (b.) Voltage modulation vs. applied field.

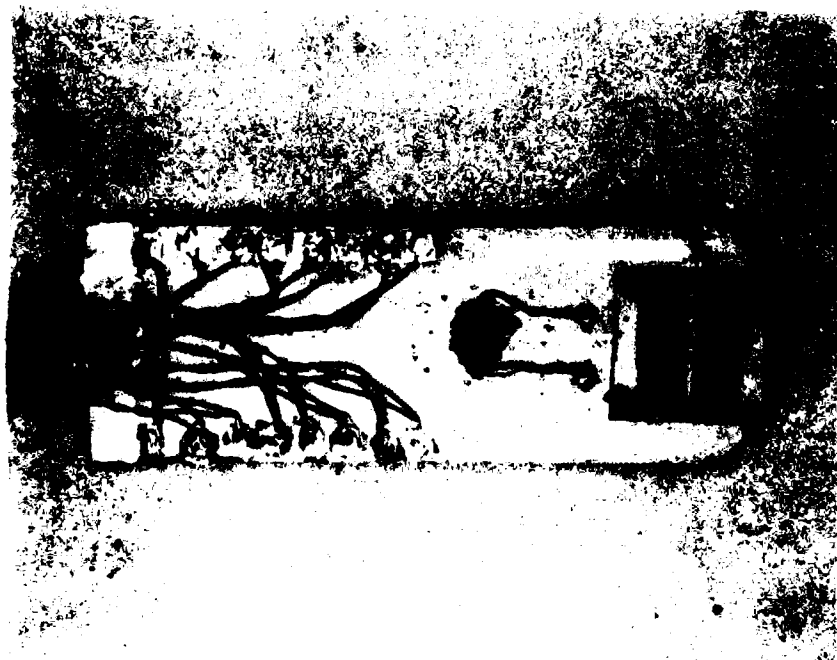


Figure 5. Photo of silicon pc board with chip attached.

1. Mounting board
2. Magnetometer chip
3. O.F.H.C. clamp
4. Solid Al end plug
5. Epoxy glass cylinder
6. Expansion space
7. Nylon displacer

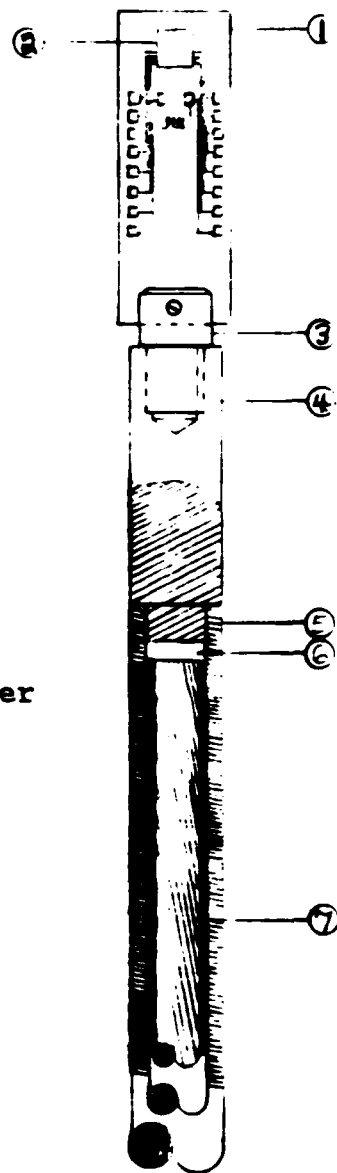


Figure 6. Sketch of the attachment of the pc board to the lower stage of the cryocooler.

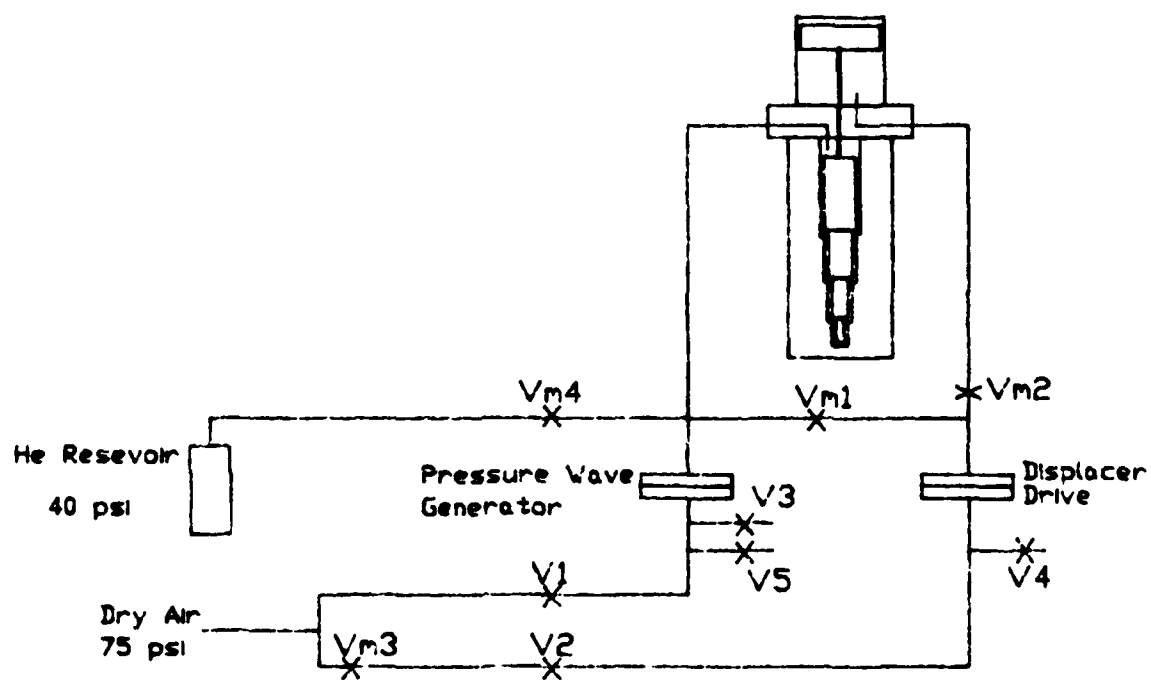
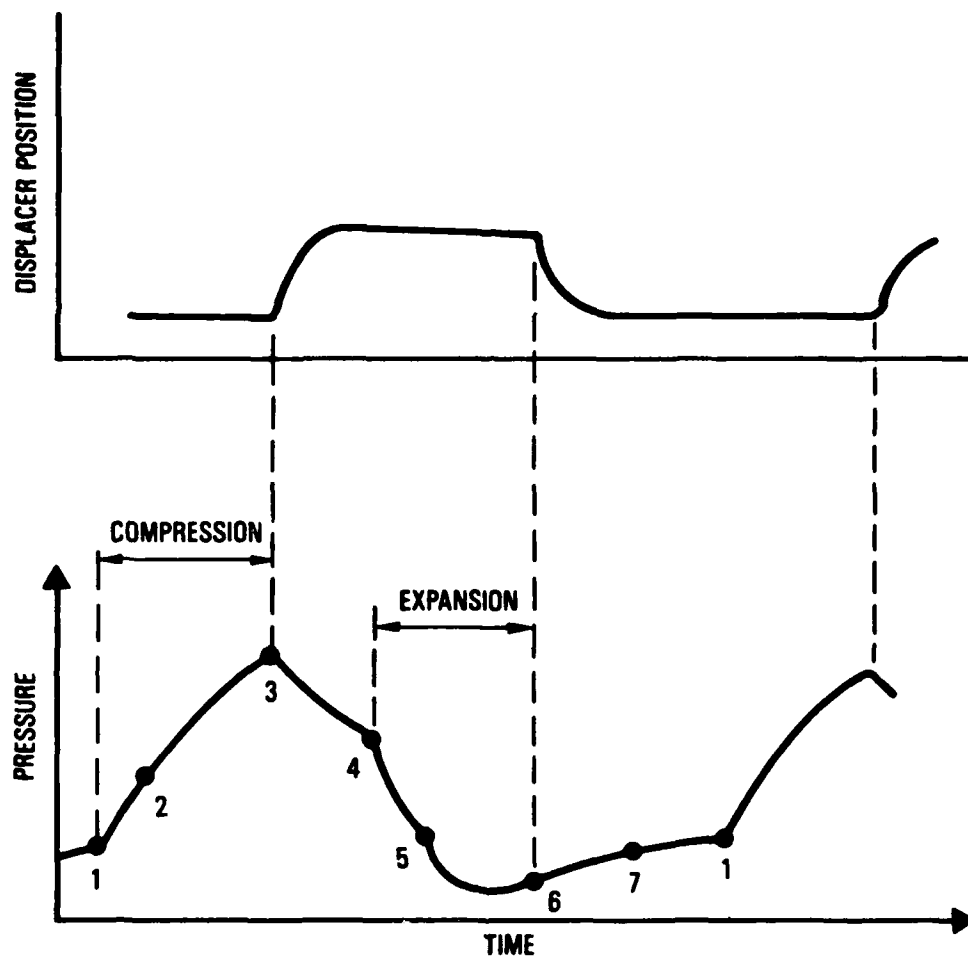


Figure 7. Schematic of cryocooler.



- |                         |                 |
|-------------------------|-----------------|
| 1. V1 OPENS             | 5. V5 OPENS     |
| 2. V4 CLOSSES           | 6. V4 OPENS     |
| 3. V2 OPENS, V1 CLOSSES | 7. V3, V5 CLOSE |
| 4. V3 OPENS, V2 CLOSSES |                 |

Figure 8. Timing diagram of the cryocooler cycle.

PAGE 1 CRYO2

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2 0          LUBES NMI AND TIMER B ON PIA #2
3 792        NMIIV EQU 80318
4 56582      TLOW EQU 80D06
5 56591      CRB EQU 8DD0F
6 56589      IRC2 EQU 8DD0D
7 65095      CONT EQU 8FE47
8 1055       SCREEN EQU 1055
9 55327      COLOR EQU 55327
10 56577      VOUT EQU 8DD01
11 53280      BORDER EQU 53280
12 49152      ORG EQU 8C000
13 49152      I
14 49152 120  SETUP      BEI
15 49153 169 80          LDA 0*START
16 49155 141 24 3        STA NMIIV
17 49158 169 192         LDA 0*START
18 49160 141 25 3        STA NMIIV+1
19 49163 169 255         LDA 8*FF
20 49165 141 3 221       STA 8DD03
21 49168 169 0           LDA 80
22 49170 141 1 221       STA VOUT
23 49173 169 1           LDA 8901
24 49175 141 192 192     STA PHASE
25 49178 141 191 192     STA TIME
26 49181          ISET UP TIMER
27 49181 173 20 200      LDA TIMERO
28 49184 141 6 221       STA TLOW
29 49187 173 21 200      LDA TIMERO+1
30 49190 141 7 221       STA TLOW+1
31 49193 169 1           LDA 8901
32 49195 141 15 221      STA CRB
33 49198 173 13 221      LDA IRC2
34 49201 9 130           ORA 8982
35 49203 141 13 221      STA IRC2
36 49206 88              CLI
37 49207 96              RTS
38 49208          ISTOP-ENTRY ---
39 49208 120            STOP
40 49209 169 0           BEI
41 49211 141 1 221       LDA 80
42 49214 141 203 192     STA VOUT
43 49217 32 156 192      JBR VIEWV
44 49220 169 71          LDA 0*CONT
45 49222 141 24 3        STA NMIIV
46 49225 169 254         LDA 0*CONT
47 49227 141 25 3        STA NMIIV+1
48 49230 88              CLI
49 49231 96              RTS
50 49232 234            START  NOP
51 49237 72             PHA
52 49234 178            TIA
53 49235 72             PHA

```

ISET UP USER PORT  
PIA#2 PORTB  
ICLOSE VALVES  
SET FOR BEGINNING  
OF CYCLE

ENABLE IRQ

ICLOSE VALVES  
IDISP VALVES

```

54 49236 173 13 221     LDA IRC2
55 49239 141 194 192     STA IRC
56 49242 41 2           AND 8902
57 49244 240 48         BEQ CONTU
58 49246 234            BEGIN  NOP
59 49247 206 191 192     DEC TIME
60 49250 208 32          BNE EXIT
61 49252 206 192 192     DEC PHASE
62 49255 16 6           BPL STEP
63 49257 173 193 192     LDA CNUM
64 49260 141 192 192     STA PHASE
65 49263 174 192 192     LDX PHASE
66 49266 189 0 200       LDA VTBL,X
67 49269 141 1 221       STA VOUT
68 49272 141 203 192     STA NMI
69 49275 189 10 200      LDA TTBL,X
70 49278 141 191 192     STA TIME
71 49281 32 156 192      JBR VIEWV
72 49284 234            EXIT   NOP
73 49285 169 130         LDA 8982
74 49287 141 13 221     STA IRC2
75 49290 104            PLA
76 49291 170            TAI
77 49292 104            PLA
78 49293 64             RTI
79 49294 173 194 192 192  CONTU  LDA IRC
80 49297 9 128           ORA 8980
81 49299 141 13 221     STA IRC2
82 49302 104            PLA
83 49303 170            TAI
84 49304 104            PLA
85 49305 76 71 254       JMP CONT
86 49308          I VIEWV
87 49308 162 8           LDX 88
88 49310 169 6           LDA 86
89 49312 141 32 208      STA BORDER
90 49315 141 33 208      STA BORDER+1
91 49318 162 5           LDX 85
92 49320 173 199 192 192  LP     LDA ON
93 49323 78 203 192      LSR MEM
94 49326 176 3           BCS NX
95 49328 173 201 192     LDA OFF
96 49331 157 21 4 192     STA SCREEN,X
97 49334 169 1           LDA 81
98 49336 157 21 216      STA COLOR,X
99 49339 202            BEI
100 49340 208 234        BNE LE
101 49342 96             RTS
102 49343          ILE
103 49344          ILE
104 49345          CNUM  BYT 809
105 49346          IRC   BYT 800
106 49347          TEMP  BYT 800000000
107 49351          ON    BYT 86
108 49353          OFF   BYT 87
109 49355          MEM   BYT 800
110 51200          EQU 8C00
111 51200          VTBL  BYT 80908B1C141604020201
112 51210          TTBL  BYT 81EFA3C82E6AE61EFA78
113 51220          TIMERO BYT 8E803 (T=1000MS)
232 3
ERRORS=0
NMIIV 792 TLOW 56582 CRB 56591 IRC2 56589 CONT 65095
SCREEN 1055 COLOR 55327 VOUT 56577 BORDER 53280 ORG 49152
SETUP 49152 STOP 49208 START 49232 BEGIN 49246
EXIT 49284 CONTU 49294 VIEWV 49308 CNUM 49345 IRC 49346
TIME 49343 PHASE 49344 NMI 49345 VTBL 51200 TTBL 51210

```

SAVE REG  
FOR THIS  
INTERRUPT  
NOT DONE EXIT  
NEXT PHASE  
CONT WITH NEXT PHASE  
RESTART CYCLE  
DISP VALVES  
RESTORE REG.  
RESTORE IRQ MASH  
BLUE BORDER AND  
LAND BACK GROUND  
WHITE CHAP

Figure 9. Assembly language listing of the control program.



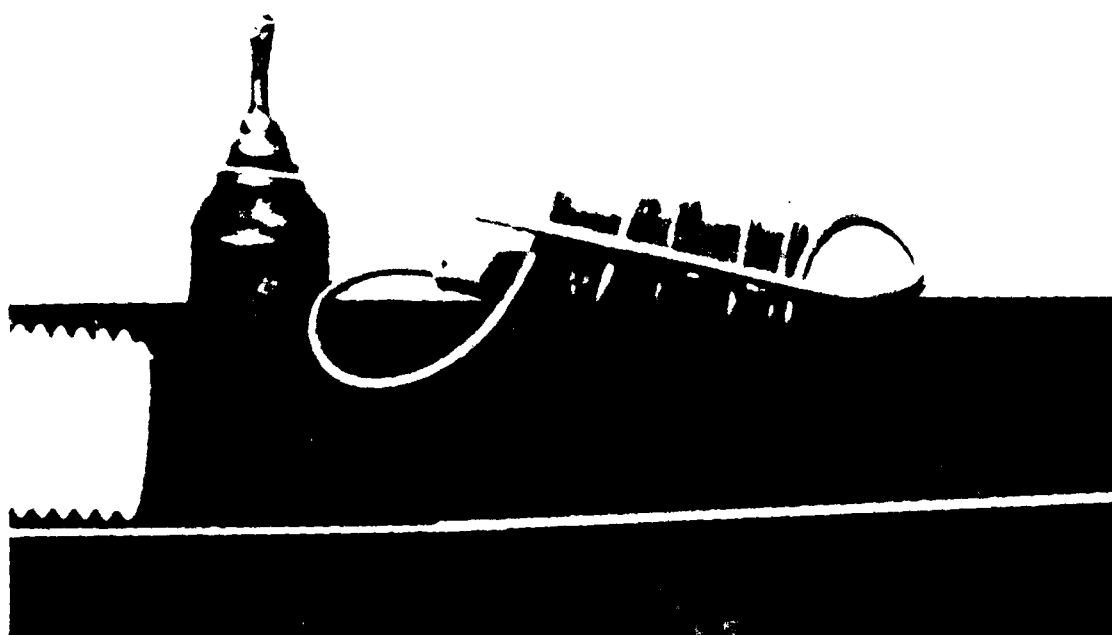


Figure 10. Photo of original (left) and new Joule-Thompson stages.





Figure 12. Photo of compressor and valving assembly.

END

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